

EUROPEAN ATOMIC ENERGY COMMUNITY - EURATOM

CODE CROOC

ORGEL REACTOR CONTAINMENT DESIGN PARAMETERS

by

R. SIMON and H. I. DE WOLDE





ORGEL Program

Joint Nuclear Research Center Ispra Establishment - Italy

ORGEL Project and Scientific Data Processing Center - CETIS

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ORGEL Program Joint Nuclear Research Center - Ispra Establishment (Italy) ORGEL Project and Scientific Data Processing Center - CETIS Luxembourg, April 1969 - 50 Pages - 13 Figures - FB 70

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The code is fitted with a CALCOMP subroutine, so that all results can be plotted immediately.

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ABSTRACT

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KEYWORDS

C-CODES TEMPERATURE PRESSURE ORGANIC COOLANT LOSSES ORGEL REACTOR

CODE CROOC

ORGEL reactor containment design parameters

Contents

Introduction

- 1. Description of the problem and calculating model
- 1.1 Containment system
- 1.2 The accident
- 1.3 Physical properties of involved fluids
- 1.4 Calculating process
- 1.5 Various possible applications
- 1.6 Example
- 2. Description of program
- 2.1 General
- 2.2 Description of the input for CROOC
- 2.3 The output
- 3. FORTRAN listing of CROOC

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Introduction *

The code CROOC has been established in order to assess the variations of pressure, temperature and concentrations in an ORGEL reactor containment due to an important leakage of organic coolant, and to study the efficiency of such consequence-limiting safeguards as dousing systems and internal separations.

The code essentially describes the thermodynamic comportment of a mixture of air or nitrogen, hot reactor coolant HB-40 and dousing water. Whereas the state of air and water can be adequately described by simple and well-known state equations and gas laws, the thermodynamics of the reactor organic coolant mixture can only be correlated in a rather approximative manner.

For closed or vented rooms of the containment builing, into which the leakage gains access, the time dependent state variables temperature and pressure are computed for arbitrary time steps from the mass and energy balances. By means of steady-state equations, the code calculates the state in the vault, in which a severe coolant discharge occurs. It also determines the effect of a rupture disk, which relieves the overpressure in the vault to the adjacent reactor hall by computing the discharge rate to the hall and the resulting rise in hall pressure, temperature and density. The intervention of dousing systems is taken into account.

1. Description of the problem and calculating model

1.1 Containment system

The present safety studies for the ORGEL prototype have resulted in a containment building concept as presented schematically in Fig. 1.

^{} Consequences of <u>Rupture</u> Of Organic Circuits**

^{*} Manuscript received on 3 January 1969.

The piping of the primary organic circuit and the reactor core are enclosed in a concrete vault C, which takes in the lower part of a cylindrical containment vessel. The two vessels are supposed to be completely separated under normal operating conditions. A number of rupture disks between the hall and the vault are provided in order to relieve extreme over-pressures in case of very severe piping failures. Both the circuit vaults and the servicing hall, which forms the major part of the remaining containment space, are provided with water dousing systems.

The water dousing system is assumed to go into action with a delay due to detection, signal transmission and release mechanism inertia. As, during this delay, the temperature of the affected rooms has risen considerably, the injected water will evaporate completely if the dousing rate is low compared with the leak rate. In this case, the rise of temperature is attenuated, whereas a reduction of the temperature can only be expected after a decrease in leakage. With at least two independent water injection systems (dousing and sprinkler in the vault, dousing, sprinkler and wall cooling in the hall), installed, at least one system in each room can be assumed to operate on schedule. The remaining systems could most probably be taken into operation at a later moment.

1.2 The accident

The hypothetical accident is assumed as follows :

a) The coolant discharge

Immediately after the rupture of a coolant pipe, at zt = 0 of the elapsed time scale, the coolant mass discharge rate will assume its maximum and remains at this value until the interventive actions upon the circuit are effective. These

interventions are the depressurization of the circuit by blowingoff the surge tank cover gas and the closing of the isolating valves situated on both sides of the primary piping penetrations of the containment.

The reduction for the leak rate due to these interventions is assimilated by an exponential decrease. Of course, time dependent functions of the leak rate that may fit the actual conditions better can be introduced as driving functions.

b) The consequences of the coolant discharge

The hot coolant leakage transfers a part of its thermal energy to the surrounding atmosphere, thereby raising its temperature and the pressure in the closed volume. Another part of the coolant energy serves to evaporate a fraction of the coolant, the vapour pressure of which adds to the pressure of the previously present gas N_2 or air to form the total pressure in the mixture.

In case of a severe piping rupture, the pressure in the vault may rise to the design pressure of the vault walls before any interventive counteractions can reasonnably be assumed to be effective. In order to avoid damage to the concrete vault, a number of rupture disks will relieve the vault pressure to the hall, when a pressure limit is exceeded.

The mixture of coolant vapour and nitrogen, which is discharged from the vault to the hall, will provoke a temperature increase and an over-pressure in the hall. In the course of the discharge, the nitrogen is gradually purged from the vault, in which the coolant vapour pressure rapidly attains saturation.

As long as the coolant leakage rate is important, the differential pressure between vault and hall is high enough to admit critical flow through the orifice. Then, as the leak rate is cut down, the flow in the hall gradually decreases and the pressure difference between the two rooms disappears.

b) The decline of the pressure and temperature

As seen as the transfer rate of energy from the coolant to the room atmosphere is surpassed by the amount of energy primarily absorbed by the water and dissipated to the environment by heat transfer and building leakage, the temperature and the pressure in the containment decrease.

As the code CROOC was written essentially to determine the time and magnitude of the peak pressures and temperature, the subsequent decline of these two state variables after their peak values is calculated in a simplified manner, taking the conservative assumptions that heat transfer and leakage to the outside are negligible.

1.3 Physical properties of involved fluids

1.3.1. The reactor coolant

The hydrogenated terphenyl coolant HB-40 was chosen by the industriel group in charge of the ORGEL Prototype preliminary design, But the code may be adapted to other coolants. To take account of the changements of compositions due to continuous irradiation and heating, a mixture containing roughly 20 % high boilers and 15 % light boilers was assumed to present the equilibrium coolant. For the liquid phase data concerned the following mean constant values were used :

> Liquid density: $S_o = 800 \text{ kg/m}^3$ Liquid heat capacity: $c_o = 2500 \text{ J/g} \circ C$

For the coolant vapour, no data except the saturated vapour pressure as function of the temperature t (°C) were available :

$$P_{osat = e} 1.18.10^{-2} (t - 244)$$
(bar)

The remaining data needed for the calculation of phase equilibrium and the mass and energy balances had to be estimated.

The following model was applied to obtain a somewhat realistic, but conservative assessment of the thermodynamic comportment of the coolant :

The coolant is composed of only two fractions, of which the lighter one has the characteristics of diphenyl, whereas the heavier fraction, having a very low fugacity, is not considered present in vapour phase.

The vapour heat capacity of the light fraction is assumed to equal the liquid heat capacity c_0 (in reality, it is probably about 20 % lower).

The mass proportion of <u>light fraction</u> total coolant

is defined as a compressibility factor K.

The partial pressure of non-saturated coolant vapour P_{og} is determined with the gas equation from the mean coolant density $(Q_{o} = \text{total coolant mass present per unit of volume})$

$$P_{og} = \frac{S_o \cdot R_o \cdot (t + 273, 15)}{K_1 \cdot 10^5}$$
 (bar)

where R_{α} is the gas constant of diphenyl

$$R_{2} = 36,8 \ J/kg \ ^{\circ}K$$

As specific heat and the normal density of air differ only slightly (2% and 3% respectively) from those of nitrogen, the mean physical properties of the hall and vault atmospheres are considered to be equal :

> Spec. heat at constant vol. $c_{vl} = 720 \text{ J/Kg} \circ C$ Gas constant $R_1 = 287 \text{ J/Kg} \circ K$

All state variables are determined from the gas equation.

1.3.3 <u>Water</u>

The pressure of the saturated vapour P_{wsat} results from the integration of the Claurius-Clapagron equation :

$$P_{hw} = e \frac{49,487 - \frac{6850}{(t+273,15)} - 5,25 \ln(t+273,15)}{(t+273,15)}$$
(bar)

The state of non-saturated vapour is computed from the gas equation with

$$Rw = 460, 0 J/Kg ^{\circ}K$$

In order to simplify the energy balance, the enthalpy of the water and the heat of evaporation were lumped to a mean heat o evaporation r_m :

$$r_{m} = 2.5.10^{6} J/Kg$$

1.3.4 Coolant-air-water mixture

For the determination of the mass discharge, it can be assumed that only a part of the coolant leakage is airborn and thus likely to be carried over to the hall once the rupture disk is open.

This fraction is defined as "mist fraction".

$$\mu = \frac{\text{mass of airborne coolant}}{\text{total mass of coolant}}$$

The density of the coolant-air-water mixture is the sum of densities of airborne coolant, water vapour and air :

$$\int g = \mu \cdot g \circ + g w + g L$$
 kg/m³

The flow of this mixture through the orifice is adiabatic, the exponent of the adiabatic expansion is assmed

$$n = 1,135$$
 (-)

as for the saturated water vapour. For this exponent, the critical pressure ratio is :

$$C_1 = 0,577$$
 (-)

1.4 Calculation process

1.4.1 General procedure

A general scheme of the calculation is presented in Fig. 2. The independent variable in the main process of calculation is the elapsed time zt, which is divided into a suitable number of time steps Δz .

The conditions at the beginning of the first time step are known. With these initial conditions, the state at the end of the time step is computed. The obtained results serve to calculate the conditions after the following time step and so on.

The step-by-step procedure is stopped at a chosen time limit.

1.4.2 Known data

a) The physical properties (s.1.3)

CVL, CO, Ri, Rw, rm, K1, n,

b) The parameters (varying with the cases) :

"Mist fraction"	M
Initial coolant leak rate	Z ₀₁
Initial dousing rate in vault	Z
Initial dousing rate in hall	Z
Time of intervention for dousing vault	Z
Time of intervention for dousing hall	z hs
Temperature of leakage	to
Vault volume	v _ĸ
Hall volume	V E
Free section of rupture disk	ø
Design pressure of rupture disk	Pb
Duration of each time step	Δz

c) Initial conditions (time zt = 0)

Density of atmosphere in vault \mathcal{S} liDensity of atmosphere in hall \mathcal{S} hl1Temperature in vault t_1 Temperature in hall t_{h1} Mass of coolant spilled $M_{01} = 0$

1.4.3. State in the vault

For the determination of the mean temperature t_{i+1} in the vault at the end of time step ΔZ_{i+1} , a simplified balance of energy gives :

$$t_{i+1} = \frac{\int_{1i} V_k \cdot C_{v1} \cdot t_i + M_{oi} \cdot C_{o} \cdot t_i + Z_{oi+1} \cdot C_o \cdot \Delta \mathbf{z} \cdot t_i - Z_{wi+1} \cdot \Delta \mathbf{z} \cdot \mathbf{r}_m}{\int_{1i} V_k \cdot C_{v1}} + (M_{oi} + \mathbf{z}_{oi+1} \cdot \Delta \mathbf{z}) \cdot C_o}$$

$$(^{\circ}K) = 1$$

Wherein the leak rate Z_{oi} and the dousing rate Z_{wi} are only functions of the elapsed time known in advance. The mass of coolant dispersed per unit of vault volume is :

$$\int_{0i+1}^{\infty} = \frac{M_{oi} + Z_{oi+1} \cdot \Delta z}{V_{k}} \qquad (\frac{(Kg)}{(m3)}) 2$$

As long as the vault atmosphere is not satured with coolant vapour, the partial pressure of the latter is given by the gas equation :

$$P_{\text{ogi+1}} = \frac{\int o_{i+1}^{+} \cdot R_{o} \cdot (t_{i+1} + 273, 15)}{K_{i} \cdot 10^{5}} \quad (bar) \quad 3$$

The atmosphere is satured when the pressure attains :

$$P_{osi+1} = e^{1.18.10^{-2}}$$
 (t - 244) (bar) 4

The partial vapour pressure of the coolant p_{oi+1} for a given temperature t is always the lower value resulting of 3 and 4.

Once the dousing has begun, the quantity of water present in the mixture at the time i+1 is :

$$M_{wi+1} = M_{wi} + Z_{wi+1} \cdot \Delta z \qquad (kg) 5$$

where

$$M_{wi} = \sum_{0}^{i} Z_{w} \Delta z \qquad (kg) 6$$

The density of the water dispersed in the vault is :

$$\int w_{i+1} = \frac{M_{wi} + Z_{wi+1} \cdot \Delta_z}{V_k} \qquad \frac{(kg)}{(m3)} = 7$$

As complete evaporation is assumed, the partial pressure of the water vapour is :

$$P_{wi+1} = \int_{1i+1} \cdot R_1 \cdot (t_{i+1} + 273) \cdot 10^{-5} (bar) 8$$

and the partial pressure of the nitrogen

$$P_{1i+1} = S_{1i+1} \cdot R_1 \cdot (t_{i+1} + 273) \cdot 10^{-5}$$
 (bar) 9

wherein

as long as the vault remains closed.

The total pressure in the vault \mathbf{P}_{vg} of the mixture of coolant, water and nitrogen adds up to

$$\mathbf{P}_{gi+1} = \mathbf{P}_{oi+1} + \mathbf{P}_{wi+1} + \mathbf{P}_{li+1}$$
 (bar) 11

and the density of the mixture is :

$$S_{\text{vgi+1}} = \mu \cdot S_{\text{oi+1}} + S_{\text{wi+1}} + f_{\text{li+1}} \qquad (\frac{\text{kg}}{\text{m}3}) \qquad 12$$

.

The mass concentrations of coolant X_{i+1} and water Y_{i+1} in the mixture are :

$$X_{i+1} = \frac{\mu \cdot S_{oi+1}}{gi+1}$$
 (-) 13

$$Y_{i+1} = \frac{9}{9} \frac{w_{i+1}}{g^{i+1}}$$
 (-) 14

To determine, whether the rupture disk has opened, the disk design pressure P_{b} is compared with the total vault pressure P_{ji+1} computed in eq. 11:

If $P_{gi+1} < P_b$; the vault remains a closed volume and the state in the hall is unchanged.

The development in the vault during the following time step i+2 can than be performed directly, as all conditions at the end of **st** i.e. Partial and total pressures, temperature, density and the concentrations are known.

1.4.4 The discharge from the vault

If $P_{gi-1} > P_b$, the rupture disk has burst and the calculation proceeds to the calculation of the discharge from the vault. The discharge of the gas-coolant-water mixture is assumed to be isentropic. The critical pressure ratio $C_1 = 0,577$ corresponds to that of satured water steam with the exponent $n = c_p/c_v=1.135$, therefore if

$$\lambda_{i+1} = P_{hgi}/P_{gi+1} \leq C_1$$

(P : total hall pressure, p : total vault pressure)
h9
The flow through the orifice is critical, and the flow coefficient is a constant :

$$\Psi_{i+1} = c_1^{\frac{1}{n}} \sqrt{\frac{n}{n-1}} (1 - c_1^{\frac{n-1}{n}}) \qquad (-) \quad 15$$

For
$$\lambda_{i+1} = P_{hi}/P_{gi+1} \langle c_1 \rangle$$

The flow is subcritical and the flow coefficient is

1-

$$\psi_{i+1} = \lambda_{i+1}^{\frac{1}{n}} \sqrt{\frac{n}{n-1}} (1 - \lambda_{i+1}^{\frac{n-1}{n}}) (-) 16$$

The mass discharge rate G through the rupture disk orifice with a section O results from the flow equation

$$G_{i+1} = 0.\psi_{i+1} \sqrt{2P_{gi+1}} \cdot g_{gi+1} \cdot 10^5$$

The partial mass discharge rates of coolant A_0 , water A_w and nitrogen A that flow into the hall are :

$$A_{oi+1} = X_{i+1} \cdot G_{i+1}$$
 (kg/s) 17

$$A_{Wi+1} = Y_{i+1} \cdot G_{i+1}$$
 (kg/s) 18

$$A_{li+1} = G_{i+1} - A_{oi+1} - A_{wi+1}$$
 (kg/s) 19

1.4.5 Influence of discharge on conditions in vault

.

Due to the discharge the masses of coolant M and water M w in the vault at the end of time step i+1 are

$$M_{oi+1} = N_{oi} + Z_{oi+1} \cdot \Delta \mathbf{z} - A_{oi+1} \cdot \Delta \mathbf{z}$$
 (Kg) 20

$$M_{wi+1} = M_{wi} + Z_{wi+1} \cdot \Delta \mathbf{z}^{-A}_{oi+1} \cdot \Delta \mathbf{z} \quad (kg) \qquad 21$$

The nitrogen density $\int 1$ is reduced to

As a considerable mass of coolant is not in gaseous phase, the expansion of the vault atmosphere is assumed isothermal. In view of the **w**ault pressure this is a conservative assumption.

1.4.6. Influence of discharge on conditions in vault

Once the rupture disk is burst, the temperature t and the total pressure $P_{\rm h}$ in the reactor hall rise due to

 - a) heat exchange with the discharged coolant-water-nitrogen mixture

- b) compression by the inflowing masses

The temperature of the hall after complete mixing and heat exchange is

$$t_{hmi+1} = \frac{(\hat{S}_{h1i} \cdot v_h \cdot c_{v1} + M_{hoi} \cdot c_o) \cdot t_{hi} + (A_{oi+1} \cdot c_o + A_{1i+1} - c_{v1}) \cdot A_z \cdot t_{i+1} - Z_{hwi} \cdot A_z \cdot r_m}{(\hat{S}_{h1i} \cdot v_h + A_{1i+1} \cdot A_z) \cdot c_{v1} + (M_{hoi} + A_{pi+1} \cdot A_z) \cdot c_o}$$

(°C) 23

The masses of coolant and water dispersed in the hall at the end of time step i+1 are :

$$M_{\text{hoi+1}} = M_{\text{hoi}} + A_{\text{oi+1}} \cdot \Delta \mathbf{z}$$
$$M_{\text{hwi+1}} = M_{\text{hwi}} + (\mathbf{z}_{\text{hwi+1}} + A_{\text{hwi+1}}) \cdot \Delta \mathbf{z}$$

The density of the hall atmosphere (nitrogen and air are lumped) for completely gaseous inflow would rise to

$$\mathbf{S}_{hi+1} = \mathbf{S}_{hi} + \frac{\mathbf{G}_{i+1} \cdot \Delta z}{\mathbf{V}_{h}} \qquad (\underline{\mathbf{kg}}) \qquad \mathbf{24}$$

However, as the coolant fraction of the discharge A is not completely gaseous and the water fraction A is rather unimportant, only the nitrogen - air density is considered in the present version :

$$9_{hli+1} = 9_{hli} + \frac{A_{li+1} \cdot \Delta z}{V_h} \qquad (\underline{kg}) \qquad 25$$

Due to the isentroptic compression the hall temperature rises to :

$$t_{hi+1} = (t_{hmi+1} + 273, 15) \left(\frac{9_{hli+1}}{9_{hli}}\right)^{0,4} - 273, 15$$

(°C) 26

The partial pressures of air P_{h1} , water P_{hw} and coolant P_{ho} in the reactor hall at the end of time step i+1 are :

$$P_{hli+1} = S_{hli+1} \cdot R_{l} (t_{hi+1} \cdot 273, 15) \cdot 10^{-5} (bar) 27$$

$$\mathbf{P}_{hwi+1} = \frac{M_{hwi+1}}{V_{h}} = \frac{M_{hwi+1}}{V_{h}} + \frac{R_{w}}{W_{hi+1}} + 273,15).10^{-5}$$
 (bar) 28

$$P_{hoi+1} = \frac{M_{hoi+1}}{K_i \cdot V_h} R_o (t_{hi+1} + 273, 15) \cdot 10^{-5}$$
 (bar) 29

or , at saturation

$$P_{hoi+1} = e^{1.18.10^{-2} (t - 244)}$$
 (bar) 30

As for the partial coolant in the vault, the calculation proceeds with the lower value obtained from eqs. 29 or 30.

At the end of time step i+1, the total pressure in the reactor hall P_{hq} has risen to

$$\mathbf{P}_{hgi+1} = \mathbf{P}_{hli+1} + \mathbf{P}_{hwi+1} + \mathbf{P}_{hoi+1}$$
 (bar) 31

This completes the calculation of the containment conditions in one time step. If the elapsed time $zt = \sum A z$ is within the predetermined time limit, the next step i+2 is calculated with the input :

$$S_{1i+1}$$
, t_{i+1} , M_{oi+1} , M_{wi+1} , M_{hoi+1} , M_{hwi+1} , S_{hci+1} , t_{hi+1}

If the elapsed time equals or exceeds the time limit, the cycle is interrupted and the calculation of the case is ended.

.

1.5 Various possible applications

1.5.1. Pressure relief containment concept

In this containment concept, the initial surge of energy released by a severe piping failure is vented off to the environment by special valves in the building. These valves are immediately shut, when the radioactivity in the building exceeds a certain limit. In this way a release of fission products to the environment is prevented without necessity of containing the entire energy of the coolant in a costly high-pressure leak-tight containment.

However, due to the uncertainty in calculating the delay between the rupture and the fission product release on one hand and the very stringent requirements as to the availability and reliability of the valve closing mechanisms on the other, this concept cannot be applied for all reactors.

In order to compute the pressures, temperatures and discharge rates for such a concept with CROOC, the following procedure is recommended.

All input data concerning the vault in the present version are replaced by the corresponding values for the reactor building. The section of the venting valves substitutes the rupture disk section and the atmosphere around the building is assimilated by taking a very high value, say 10^{10} m³, for the "hall" volume.

The action of the valves is introduced by changing 2 or 3 statements in the code in such a manner, that the discharge section is initially open and closes at a certain elapsed time.

1.5.2. Leakage to the reactor hall

A failure of the fuel handling machine or an upper channel seal are the only imaginable accidents, that would provoke a_{ii} direct leakage of organic coolant to the reactor hall.

This case can be computed $b_{\mathbf{y}}$ substituting the input data assigned to the vault. i.e volume and dousing rate by the respective values of the reactor hall and vice versa.

1.5.3. Other cases

A number of other applications may be imagined, which are more or less variations of the original problem, such as the calculation of the pressure difference between two chambers of the vault. Allthough CROOC was only intended to treat very specific problems of an ORGEL Prototype, it may readily be adapted to similar cases of a gas-cooled reactor.

1.6 Example

The present preliminary design of an ORGEL 250 Mwe Prototype is provided with a leak-tight containment. The reactor, the auxiliary coolant circuits and the main coolant piping are installed in an inertized vault of 4500 m3 free volume. It is assumed, that one dousing system will inject 167 kg/s water into the vault during at least 30 secs beginning 3 secs after the accident. The reactor servicing hall and the other locals of the building total 38.000 m^3 . The reactor servicing hall is provided with a powerfull dousing system, that also goes into action 3 secs after the accident, delivering 800 kg/s continously. A rupture disk between vault and hall with a section of 7 m^2 relieves the vault pressure, as soon as it attains 2 bar. The initial leak rate for a complete rupture of the 600 cm inner diameter coolant outlet line has been assessed to 6680 kg/s. It is assumed, that the circuit pressure and the coolant vapour pressure maintain this discharge rate during 12 secs. After this delay the venting of the circuit and the closing of the main isolation valves produce an exponential decrease of the leak rate with a period of 1 sec.

A time step length of $\Delta z_1 = 0,02$ sec for the cycles before the opening of the rupture disk, and a $\Delta_2 = 0,2$ secs for the remaining time were chosen.

The total computed time is chosen to 25 secs.

The resulting time dependent values of partial coolant, water and air (nitrogen) as will as for the total pressures and temperatures in the vault and the reactor hall are presented in Fig. 3-14. The peak values for the vault pressure and temperature are

 $P_{x max} = 2.173$ bar abs at . elapsed time zt = 12.64 secs.

and

 $t_{v max} = 333.15 \,^{\circ}C$ at zt = 2,84 secs

The corresponding values for the reactor hall are

 $P_{h max} = 2.13$ bar at zt = 21.84 secs and

 $t_{h max} = 113,39$ °C at zt = 13.04 secs

It should be noted, that, as the code has not yet been adapted to treat the state of saturated and condensing steam, the calculation of the partial water pressure; in the hall is slightly incorrect from the 19 th. second onward. As at this moment the increase of partial water pressure cannot continue beyond the saturation value, while the temperature decreases, the actual **peak** pressure is obtained, when the steam is saturated at P_{w} sat = 0.615 bar and t_w sat = 86.7 °C (zt = 18.84 secs). The subsequent decrease of temperature and pressure has been determined by hand calculation (s. dotted line Fig.11)

2. The program CROOC

2.1 General

The program CROOC performs the calculations, as described in part 1 of this report, and may draw graphs of the calculated results. CROOC is a FORTRAN-4 program, written for the I.B.M. 360/65 combined with a CALCOMP PLOTTER, as used at the EURATOM centre in ISPRA, ITALY. However, by removing the last part of the program i.c. the drawing of the graphs, the program may be used on other FORTRAN adapted computers also, The presented version needs about 32.000 storage places. A properly shortened version, without graph drawing, will do with about 5000 storage places as the biggest part of the mecessary memory is used for storing of the calculated results. The program may calculate up to 10 different cases in one run. Neither the number of requested graphs nor the number of curves in each graph is limited. CROOC is self-resetting, that is for each case to be calculated, the input parameters are given a defined value and only the non-standard values have to be specified in the input.

The standard values are given in the next table.

	Input	parameters and standard value for CRC	000
Par. FORTRAN Number name		Description	Standard value
1	VOLV	The volume of the vault	3800 m ³
2	OPSECV	Initial coolant leak rate	7500 kg/sec
3	TEMORG	Temperature of the coolant	360 °C
4	HU	Mist fraction	1
5	WPSECV	Initial dousing rate in the vault	167 kg/sec
6	ZS	Intervention time for dousing in	3 sec.
7	DELZ	the vault Initial tim <u>e</u> steps <u>/</u> u nti l rupture	0.02 sec
8	РВ	disk opens_/ Design pressure of rupture disk	2 bar
9	DELZA	Time steps after opening of rupture	0.2 sec
10	ZMAX	disk Total time	20 sec
11	OSURF	Free section of rupture disk	$7 m^2$
12	VOLH	Volume of the hall	38000 m ³
13	WPSECH	Initial dousing rate in the hall	149 kg/sec
14	ZSH	Intervention time for dousing in	3 sec
15	Z02	the hall Leak interruption time	10 sec
16	ZOHALF	Half time of leak interruption	1 sec
17	ZWV2	Dousing interruption time in vault	30 sec
18	WPSV2	Second dousing rate in vault	0 kg/sec
19	ZWH2	Dousing interruption time in hall	30 sec
20	WPSH2	Second dousing rate in hall	0 kg/sec
		1	

•

Some of these parameters may need some extraplanation :

The time delay in dousing is counted in seconds from the moment that the leakage starts. The dousing in the vault and the hall may start at different times. The sprinklers in the hall will not start before the pressure cap is opened. Thus, if the dousing time delay for the hall is smaller than the time at which the rupture disk opens, the sprinklers in the hall start immediately after the opening of the disk. The program has also an option for changing the dousing rates, respectively in the vault and in the hall, after a certain time (parameters 16, 17, 18 and 19). The leak may be shut-off after a certain time, according to the exponential function :

$$L_{T} = L_{i} * \sqrt{0.5} \frac{T-T_{1}}{T} \sqrt{T_{7}T_{1}}$$

in which L_{T} = leak rate at time T L; = initial leak rate T₄ = leak interrupt time t. = half time of leak shut-off

A second accident might be introduced by the specification of a negative half time.

The values of a number of physical constants are defined at the beginning of the program, one may change these constants by changing the concerning statements. The present value are given in the next table.

Symbol	FORTRAN Name	Ph ys ical constant	Value
P _{li}	RHOIN	Initial density of the air	1.2 kg/m ³
c _{vl}	CAIR	Heat capacity of air	720 j ∕g °C
C	CORG	Heat capacity of organic	2500 J /g °C
с,	CWAT	Heat capacity of water	2100 J/g °C
r m	RW	Specific evaporation heat of water	2.5*10 ⁶ j/kg
R	GASW	Gas constant of steam	469.4 J/kg °K
R	GA SA	Gas constant of air	297.5 J/kg °K
R	GA SO	Gas constant fo diphenyl	36.8 J/kg °K
К 1	COMPRO	Compressibility of organic	2
	C1	Critical pressure	0.577 bar
n	EN	Exponent of adiabatic expansion	1.135.
	ļ		

The initial values of some physical variables are defined in the programparagraph "INITIAL CONDITIONS". These values are reset for each new case.

Symbol	FORTRAN Name	Physical variable	Initial value
t _i fli t _i fli Moi Mwi Mhoi Mhoi Mhoi	ZT TEMPV RHO TEMPH RHOH QORGV QWATV QORGH QWATH	TIME Temperature in vault Density of air in vault Temperature in hall Density of air in hall Organic in vault Water in vault Organic in hall Water in hall	O SEC 20 °C 1.2 kg/m ³ 20 °C 1.2 kg/m ³ O kg O kg O kg O kg O kg

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2.2 Description of the input for CROOC

The input for CROOC consists of a combination of fixed point-, floating point-4 and alphameric information. Each element of information is limited in its actual size by the length of its field. A field exists out of a certain number of columns in a punch card. The length of the fields is defined by the format statement in the program.

A fixed point number is written without a decimal point in the utmost right part of the field.

A floating point number is written with a decimal point and eventually with an exponent. The place within the field is not important, only the exponent must be written to the utmost right. Alphamerical information might contain all the currently used symbols.

Туре	Example	Value _	Format symbol
Fixed point Fixed point Floating point Floating point Alphamerical	3 3. 3. E+1 *P = 2.0	3 30 3 30 -	I I *) E E A

*) The blanks after the first significant digit are assumed to be zeros.

The input for the program CROOC consists of two main parts :

- 1. The non-standard specifications for the cases to be calculated
- The specifications for the graphs to be drawn, including titles etc.

The program performs first all the calculations, prints and stores the results and continues with the drawing of the graphs for which a choice may be made out of all the previously calculated data.

One case to be calculated is represented by one or more cards, of which the last one has an asterisk in the first column. In each of these cards are up to 4 parameters defined which do not have standard values as mentioned in the table. The format of these cards is : $\overline{\ A2}$, I4, E12.4, 3(I6, E12.4) $\overline{\ 7}$. The program stops the actual calculations and moves to the second part if it meets a card with two asterisks in the first column. In this card the total number of graphs to be drawn is mentioned also.

There must be two or more cards for each graph, of which the last one has an asterisk in the first columns. The first introductory card of a graph-set contains the number of calculated variable which has to be plotted versus time. The columns 7-72 of this card may contain alphamerical information which will be written underneath the graph. The numbers of the variables are :

- 2 Partial pressure of organic in the vault
- 3 Partial pressure of water in the vault
- 4 Partial pressure of air in the vault
- 5 Total pressure in the vault
- 6 Temperature in the vault
- 7 Organic material in the vault
- 8 Partial pressure of organic in the hall
- 9 Partial pressure of water in the hall
- 10 Partial pressure of air in the hall
- 11 Total pressure in the hall
- 12 Temperature in the hall
- 13 Organic material in the hall

A specific card, following the first introductory card of a graph-set, must be given for each curve to be drawn. Such a card contains the number of the calculation case out of which the values for the already specified variable must be taken, to be plotted versus time.

The columns 7-72 may contain alphamerical information which will be written at the last point of the curve. It is advisable to keep these texts rather short so the graphs will not be extended too much and might be reproduced on normal report format. All the cards of a graph-set are read by CROOC with the format (A2, I4, 16A4).

The input example will illustrate the description of the data deck. The example exists out of 3 calculations and 2 graphs. The first calculation uses the standard values of the input parameters, as given on page 23, except :

The leak rate is 750 kg/sec in place of 7500 kg/sec
 The rupture disk opens at 4 bar in place of 2 bar
 The total time of interest is only 10 sec.

The second and the third case performs the calculations at different leak rates.

The first graph will contain 3 pressure curves because the number 5 means the calculated total pressure in the vault according to page 27, completed with a caption $\sum CARD 6_7$ and curve description. The second graph will only use the 12 th variable, i.c. the temperature in the hall, out of the second and the third case.

2.3 The output of CROOC

The output of CROOC consists of :

- The values of the input parameters _ page 23_7. The non-standard values are marked with an asterisk.
- The tables with the calculated variables as mentioned on page 27 ., completed with time and quantities of organic, respectively in the vault and in the hall.
- A printed message after properly finishing the preparation of the calcomp tape.
- A magnetic tape which is used as intermediary between the I.B.M. 360/65 and the calcomp equipment.

2.4 A shortened version of CROOC

As the presented version of CROOC can be used only by a very special equipment as available at CETIS-EURATOM, CROOC may easily be altered for use on other FORTRAN adapted computers. The second part of the program, i.e. the drawing of the graphs, has to be removed. In this case, the preparing of the shortened version is performed as follows :

- Remove the last part of the program which starts with : 300 NG = II(1)
- Replace this part by :

```
300 continue
```

```
stop
```

```
end
```

- 3. Remove out of the dimension statement the matrix amatr(10,14,200)
- 4. Remove the two parts of the program where this matrix is filled.

Input example for CROOC

CETIS/CADI (EURATOM)

	PROBLEM	DATE	PAGE	OF
	2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39	40 41 42 43 44 45 46 47 48 49 50 51 52 53 5	4 55 56 57 58 59 60 61 62 63 64 65 66 67	68 69 70 71 72 73 74 75 76 77 78 79 80
1				
CASE 1	2 750. 8 4.			
(2	10 10.			
CASE 2 3	2 5000.			
CASE 3 4	2 5 . E + 2			
5	* 2			
6	5 FIG. 1 PRESSURE CURVES FØR T	HE VAULT AT SØI	TE LEAK RATES.	
- GRAPH 1 {	1750 KG/SEC LEAK RATE			
8	35000 KG/SEC LEAK RATE			
6	2500 KG/SEC LEAK RATE			
(10	6 FIG. 2 TEMPERATURE CURVES FØ	RTHEHALL		
GRAPH 2 {11	25000 KG/SEC LEAK RATE			
[12	3 JOO KG/SEC LEAK RATE			
13				
. 14				
15			+++++++++++++++++++++++++++++++++++++++	
16				
12	┥╧┊┊╡┥╆┼┼┽╾┝╊╎┼┿┽╎╈┼┽┾┝╊┽┽┼╪┝┫┽┼╈┿┽╋┿┶┥			
18	┤┤┝┲┼┲┼┼┼┼┟╋┥┽┿┽┽┿┿┿┿╋╣┝┼┼┝┝╋┟┽┿┥┾╋┼┼╸			
19	┊╶┊╴╸╴╸╸╸╸╸╸╸╸╸╸╸╸╸╸╸╸╸╸╸╸╸╸╸╸			
20				
21				
22				
23	╧╪┶╶┶╾┥┫╎┥╎╴┲╼╋╍╸┽╎╴╎╼╕╴┟╌┿╼┿╼┥┥┫╴╎╴┶╼┿┻╋┥┿┱┿┿╋╋┿		+++++++++++++++++++++++++++++++++++++++	
24	<u>┿</u> ╅┽┽╎╏╎╎╷┾┽╎╬╬╎╎╎┽┢┽╎╝┿┼╎┨┾╎┼┼┿╊┾┿┽┾┼╊┾┿┥			
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FIG. 10 PARTIAL AIR PRESSURE IN HALL









FIG.14 MASS OF COOLANT IN HALL

	С С	PROGRAM CRODE BY HERMAN I. DE MOLDE AND R. SIMON Date July 1963.
	UUUU	CRODE CALCULATES THE BUILD-UP OF CONTAIMENT PRESSURE AFTER SEVERE 1PIPING FAILURE
	C	DIMENSION TIME(200), ATR(200), IC(10), ALFAB(16) DIMENSION AMATR(10,14,200), TT(3), PP(3), TTP(3), AMASS(3), ALFA(10,16) 1, AL(16)
		EQUIVALENCE (PAR(1), V)LV), (PAR(2), OPSEC), (PAR(3), TEMORG), (PAR(4), H IU), (PAR(5), VPSECV), (PAR(6), ZS), (PAR(7), DELZ), (PAR(8), PB), (PAR(9), D ZELZA), (PAR(10), ZMAX), (PAR(11), OSURE), (PAR(12), VOLH), (PAR(13), WPSEC
•		EQUIVALENCE (PAR(2),OPSECV) EQUIVALENCE (PAR(15),ZO2),(PAR(16),ZOHALF),(PAR(17),ZWV2),(PAR(18) 1,WPSV2),(PAR(19),Z'H2),(PAR(20),WPSH2) DATA STAR,STAR,DSTAR,DINK(/**,***,***,***,***,***
	C—	DATA TT, PP, TTP, AMASSZO.0, 0.00001, 10.0, 5.0, 0.0, 0.0, 400.0, 0.0, 0.0, 1. 12E+5, 0.0, 0.0Z
	Č	RHDIN=1.2 CAIR=720. CORG=2500.
	ſ	CWAT=2100 RW=2.55+6 GASW=400.4 GASW=400.4 GASA=297.5 GASD=36.3 COMPRO=2.0 C1=0.577 EN=1.135 ICASE=0
	Č-	FIXED VALUES FOR THE VARIABLE INPUT PARAMETERS
		80 CONTINUE 1 VOLV=3300. 2 OPSECV=7500. 3 TEHDRG=360. 4 HU=1. 5 WPSECV=167. 6 ZS=3. 7 DELZ=0.02

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42

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11 OSURF= 7. 12 VULH=38000. 13 WPSECH=149. 14 ZSH=3. 15 Z02=10. 16 ZÜHALF=1. $17 \ ZwV2=30$. 18 WPSV2=0. 19 ZWH2=30. 20 WPSH2=0. DU 89 I=1,20 89 ST(I)=BLNK С --READ INPUT-C. 90 REAU(5,92)TAR,(II(I),PARM(I),I=1,4) 92 FURMAT(A2,I4,E12.4,3(I6,E12.4)) DU 96 I = 1, 4IF(II(I))96,96,94 94 IA=II(I) ST(IA)=STAR PAR(IA) = PARM(I)90 CONTINUE IF(TAR.EQ.STAR)GO TO 98 IF(TAR.NE.DSTAR)GO TO 90 GU TO 300 98 CONTINUÉ С Č-----INITIAL CUNDITIONS------ZT=0. TEMPV=20. QURGV=0. NCYCL=0 NWAY = 1RHJ=RHUIN WSWIT=0. QWATV=0. PH=1. WSWITH=0. RH0H=1.2 QURGH=0. TEMPH=20. QMATH=0. DZ=DELZ PH0=0. PHW=0. PHA=RHUH*GASA*(TEMPH+273.)*1.E-5 PH=PH0+PHW+PHA G=0.

43

C	TT(3) ICASE	AX ASE+1	
Č Č	WRITE	PJT	
L	UPSA= TAU=A ⊿RITE	ECV (0.5)/(-ZOHALF) 101)	
]	101 FORMA WRITE 1ST(5) 2T(11)	1H1/' ***** INPUT DATA *****'///) 102) VOLV,ST(1),OPSECV,ST(2),TEMORG,ST(3),HU,ST(4),WPSEC ,ST(6),DEL2,ST(7),PB,ST(3),DELZA,ST(9),ZMAX,ST(10),OSURF LH,ST(12),WPSECH,ST(13),ZSH,ST(14)	V,
j.	LO2 FURMA A' 1' B' 2' C' 3'	VOLUME OF VAULT, 14X, F12.3, M3 ,A1// LEAK RATE, 20X, F12.3, KG/SEC ,A1// TEMPERATURE OF ORGANIC, 7X, F12.3, DEGREE C ,A1//	
	し、 4、 5、 5、 6、 7、 3、 3、 3、 3、 5、 1、 5 1、 5 1、 5 1、 5 1、 5 1、 5 1 5 5 5 1	MIST,FRACTION',16X,F12.3, ',A1// DOUSING RATE IN VAULT',8X,F12.3, 'KG/SEC ',A1// TIME DELAY DOUSING IN VAULT',2X,F12.3, 'SEC ',A1// TIME STEP 1',13X,F12.3, 'SEC ',A1// PUDTURE DISK ODENS AT BY E12.3 'PAP 'A1//	
	I 9' J 10' K 11'	TIME STEP 2',18X,F12.3, SEC ',A1// TOTAL TIME ',18X,F12.3, SEC ',A1// FREE SECTION OF RUPTURE DISK',1X,F12.3, M2 ',A1//	
	M• 13• N• 14• WRITE ±•ST(1)	DUUSING RATE IN HALL',9X,F12.3,' KG/SEC ',A1// TIME DELAY DUUSING IN HALL',3X,F12.3,' SEC ',A1/) 105)202,ST(15),ZUHALF,ST(16),ZWV2,ST(17),WPSV2,ST(18),ZW WPSH2.5T(20)	√H2
]	105 FÜRMÄ A' 15' B' 15'	LEAK INTERRUPT TIME ,10X,F12.3, SEC ',A1// HALE TIME',20X,F12.3, SEC ',A1// DUUSING IN VALUE INTERRUPTED ,1X,E12.3, SEC ',A1//	
	0* 18* E* 19* F* 20*	SECOND DOUSING RATE IN VAULT, 1X, F12.3, KG/SEC ,A1// DOUSING IN HALL INTERRUPTED, 2X, F12.3, SEC ,A1// SECOND DOUSING RATE IN HALL, 2X, F12.3, KG/SEC ,A1//)	
1	LO3 FORMA	INT STEP TIME RELATIVE PRESSURES IN VAULT TEMP. ", IC ",10X, IC ",10X,	
1	2* REL 312X, 4* OR LO4 FORMA	WATER AIR TOTAL ',8X,' IN VAULT',10X, WATER AIR TOTAL ',8X,' IN VAULT',10X, WATER AIR TOTAL ',8X,' IN HALL DISCHARGE'//) I4,F6.2,4F7.3,F7.2,E11.3,10X,4F7.3,F7.2,2E11.3)	
_	PVA=RI PV()=0 PVw=0 PVw=0	GASA*(TEMPV+273.)*1.E-5	
C	Số Tổ	0	

.

C

C		-CALCULATIONS FOR THE VAULT
С		
	110	ARG1=RHQ*VQLV*CAIR*TEMPV
		7T=7T+DEL7
	1 / 0	
	140	1F(ZWV2+G)+Z1)G0 10 142
		WPSECV=WPSV2
	142	IF(ZWH2.GT.ZT)G0 T0 144
		WPSECH=WPSH2
	144	CINTINIE
	T 1 1	
	1 2 5	
	125	
		ARGZ=QURGV*CURG*TEMPV
		ARG3=DPSECV*DELZ*TEMORG*CURG
		ARG4=∀PSECV≠DELZ*R₩÷₩S₩IT
		$ARG5 = RHD \neq VDLV \neq CATR$
		ARG6 = (0)RGV + 0PSECV + 0EL7) + CORG
		TEMPV = (ARC1 + ARC2 + ARC3 - ARC3 - ARC4) / (ARC5 + ARC6)
		NCVCI-NCVCI-1
	112	
	112	PVU=RHUVU*GASU*(1EMPV+2/3.15)*1.E-5/CUMPRU
		PVQX=EXP(0.0118*(TEMPV-244.))
		IF(PVOX.GT.PVO)GO TO 113
	113	CONTINUE
		PVA=RHU+GASA*(TEMPV+2/3·)*I·E=5
		Ρν=Ρνυ+Ρνη+Ρνα
		RHQVG=HU*RHQV0+RHQVW+RHQ
		X=(HJ*RH0V9)/RH0VG
		Y=RHDVW/RHUVG
C.		
č		
Ŭ		GU TU- (126-200) - NWAY
	1 2.	10 + 12 + 12 + 12 + 12 + 12 + 12 + 12 +
	120	
	120	
	130	
-		WRITE(6,104)NCYCL,ZT,PVU,PVW,PVA,PV,TEMPV,QURGV
Č		
C-		-STORE THE CALCULATED RESULTS
С		
		I=ICASE
		J=NCYCI +1
		$\overline{A}M\overline{ATR}(\overline{I},\overline{I},L) = 7T$
		$\Delta M \Delta T R (T = 2 - 1) = \overline{D} V \Omega$

.

```
AMATR(I,4,J)=PVA
      AMATR(I, 5, J) = PV
      AMATR(I,6,J)=TEMPV
      AMATR(I,7,J)=QORGV
      AMATR(I, \partial, J) = PHO
      AMATR(I,9,J) = PHW
      AMATR(I, 10, J) = PHA
      AMATR(I, 11, J) = PH
      AMATR(I,12,J)=TEMPH
      AMATR(I, 13, J) = QORGH
      \LambdaMATR(I,14,J)=G
      JURGV= DORGV+DELZ*OPSECV
      GU TU 110
  132 HWAY=2
      GU TO 200
  134 STUP
С
  C
  200 AMDA=PH/PV
      IF(AMDA.LT.1.)GD TO 203
      WRITE(6,201)
  201 FURMAT (* END OF RUN BY P-VAULT IS LESS THAN P-HALL !)
      ZMAX=ZT-DELZ
      GU TU 223
  203 CONTINUE
      IF(ZSH.GT.ZT) GO TO 202
      WSWITH=1.
  202 IF(AMDA-C1)204,204,206
204 PSI=(C1**(1./EN))*SQRT(EN*(1.-C1**((EN-1.)/EN))/(EN-1.))
      GU TU 208
  206 PSI=ĀdDA**(1./EN)*SQRT(EN*(1.-AMDA**((EN-1.)/EN))/(EN-1.))
  208 CONTINUE
      G=OSURF*PSI*SQRT(2.*PV*RHOVG*1.E+5)
      AU=X÷G
      AN=Y*G
      AA=G-AO-AW
С
С
 -----QUANTITY OF ORGANIC AND LOSS OF WATER IN THE VAULT------
      QORGV= JORGV+DELZ*OPSECV-A0*DELZ
      QWATV = (QWATV - AW * DZ) * WSWITH
      RHU=RHO-AA*DZ/VOLV
      IF(ZSH.GT.ZT) GO TU 220
      WSWITH=1.
  220 ARG1=(RHOH*VULH*CAIR+OORGH*CORG)*TEMPH
      QWATH=QWATH+AW*DZ*WSWIT+WPSECH*DZ*WSWITH
      ARG7=QWATH*CWAT*TEMPH
      ARG8=QWATH≉CWAT
      ARG2=(AO*CURG+AA*CAIR)*DZ*TEMPV
```

```
ARG3=(WPSECH*DZ*RW)*WSWITH
    ARG4=(RHOH*VOLH+AA*DZ)*CAIR
    ARG5=(QURGH+AU*DZ)*CORG
    TEMPH=(ARG1+ARG2-ARG3+ARG7)/(ARG4+ARG5+ARG8)
    QURGH=QURGH+AU*DZ
    RHUHA=RHOH+AA*DZ/VOLH
    TEMPH=(TEMPH+273.)*(RHOHA/RHOH)**0.4-273.
    RHOH=RHOHA
    PHA=RHJH*GASA*(TEMPH+273.)*1.E-5
    PHW=QWATH*GASW*(TEMPH+273.15)*1.E-5/VOLH
    PHO = EXP(0 \cdot 0118 \times (TEMPH - 244 \cdot))
    PHOX=( )0RGH*GASO*(TEMPH+273.15)*1.E-5)/(VOLH*COMPRO)
    IF(PHUX.GT.PHO)GO TO 221
    XCH9=0H9
221 CONTINUE
    PH=PHA+PHW+PHU
    DELZ=DELZA
    DZ=DELZ
    IF((NCYCL-(50*(NCYCL/50))).NE.0)GD TO 222
    WRITE(6,103)
222 CUNTINUE
    WRITE(0,104)NCYCL,ZT, PVU, PVW, PVA, PV, TEMPV, QORGV,
   1PHO, PHW, PHA, PH, TEMPH, QORGH, G
 ---STORE THE CALCULATED RESULTS------
    I=ICASE
    J=NCYCL+1
    \overline{AMATR(I, I, J)} = ZT
    AMATR(I,2,J)=PVU
    AMATR(I,3,J) = PVW
    AHATR(I,4,J) = PYA
    AMATR(I,5,J) = PV
    AMATR(I,6,J)=TEMPV
    AMATR(I,7,J)=00RGV
    AMATR(I, 8, J) = PHO
    AMATR(I,9,J) = PHW
    AMATR(I,10,J)=PHA
    AMATR(I, 11, J) = PH
    AMATR(I, 12, J) = TEMPH
    AMATR(I, 13, J) = OURGH
    AMATR(1, 14, J) = G
    IF(ZT.LT.ZMAX)GO TO 110
223 NCY(ICASE)=NCYCL
    ZMC(ICASE) = ZMAX
    GO TO 30
----DESIGN THE GRAPHS------
300 \ (10 = II(1))
```

;

```
ING=0
302 ING=ING+1
    IF([NG-NG)300,306,380
306 CONTINUE
310 FURMAT (A2, I4, 16A4)
308 READ (5,310) A4,NV,(AL(I),I=1,16)
    100 = 0
312 ICC=ICC+1
    READ (5,312) AA, IC(ICC), (ALFA(ICC,I), I=1,16)
    IF(AA.NE.STAR)GU TO 312
    NC=ICC
    TT(3) = 0.
    DU 313 I=1,NC
    KK = IC(I)
    IF(TT(3).GT.ZMC(KK))GO TO 313
    TT(3) = ZMC(KK)
313 CONTINUE
    CUNVX=TT(3)/15.
    CALL FINIM(0.,0.)
    IF((ING-5*(ING/5)).NE.0)GU TO 314
    CALL FINIM(25 \cdot -60 \cdot)
314 CALL SYMBL4(0.0,1.0,0.3,0.0,AL,64)
    CALL FINIM(0.0, 3.0)
    IF(NV.GT.5)G0 TO 320
318 CALL DESSIN(TT, PP, 3, 1, 1, 1, 0, 0, 15, 0, 10, 0, 0, 0, 13H TIME IN SEC., -13, 1
   16H PRESSURE IN BAR, 16,0)
    CUNVY=PP(1)/10.
    GU TO 350
320 IF(NV.GT.6)GD_TD_324
321 CALL DESSIN(TT, TTP, 3, 1, 1, 1, 0, 0, 15.0, 10.0, 0, 0, 13H TIME IN SEC., -13,
   118H TEMP. IN DEGREE C, 18,0)
    CONVY=TTP(1)/10.
30 TO 350
324 IF(NV.GT.7)GU TO 326
325 CALL DESSIN(TT, AMASS, 3, 1, 1, 1, 0, 0, 15.0, 10.0, 0, 0, 13H TIME IN SEC., -1
   13,3H KG,3,0)
    CÜNVY=AMAŠJ(1)/10.
    GU TO 350
325 IF(NV.GT.11)GD TO 330
    GJ TJ 318
330 IF(NV.GT.12)GO TO 334
    GO TU 321
334 IF(NV.GT.13)GU TO 340
    GU TO 325
340 IF(NV.GT.14)GU TO 346
342 CALL DESSIN(TT, AMASS, 3, 1, 1, 1, 0, 0, 15, 0, 10, 0, 0, 0, 13H TIME IN SEC., -1
   13,7H KG/SEC,7,0)
    CUNVY=AMASS(1)/10.
    GU TO 350
340 WRITE (6,348)
```

```
348 FORMAT ( * ERROR EXIT NV IS LARGER THAN 14*)
    STOP
350 CONTINUE
    DO 364 INC=1,NC
    KC = IC(INC)
    NCYCL=NCY(KC)
    DO 360 I=1,NCYCL
    TIME(I) = AMATR(KC, 1, I)
360 \text{ ATR}(I) = \text{AMATR}(KC, NV, I)
DU 362 I=1, 16
362 ALFAB(I)=ALFA(INC, I)
    CALL DESSIN(TIME, ATR, NCYCL, 1, 1, 1, 0, 0, 15, 0, -10, 0, 0, 0, 1H, -1, 1H, 1, 0
   1)
    AM=TIME(NCYCL)
    AN=ATR(NCYCL)
    AM=AM/CONVX
    AN=AN/CONVY
    CALL SYMBL4(AM, AN, 0.15, 0.0, ALFAB, 64)
364 CONTINUE
    CALL FINIM(0.0,12.0)
    GU TU 302
380 CALL FINTRA
    WRITE(6, 382)NG
382 FURMAT (1H1/" CALCOMP HAS DRAWN', 14, " GRAPHS. ")
    STOP
                 .
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END

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Alfred Nobel

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